

# CSMD: ASSIGNMENT 10: DYNAMICS

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## Question 1

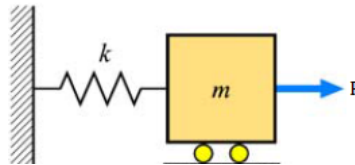


Figure 1

The system depicted in figure 1 can be modeled as:

$$ku - m \frac{d^2u}{dt^2} = F$$

The effects of  $F$  can be seen in figure 2, which has been obtained for  $k = 2, m = 1$ .  $f_n$  is defined as  $\sqrt{\frac{k}{m}}$ . As can be seen, when a constant force is applied, the system oscillates with the natural frequency  $\frac{\sqrt{f_n}}{2\pi} = 0.225$  (a whole cycle every 4.44 s). However, for a sinusoidal load with the natural frequency of the structure, the result is unbounded and tends to increase its amplitude without control.

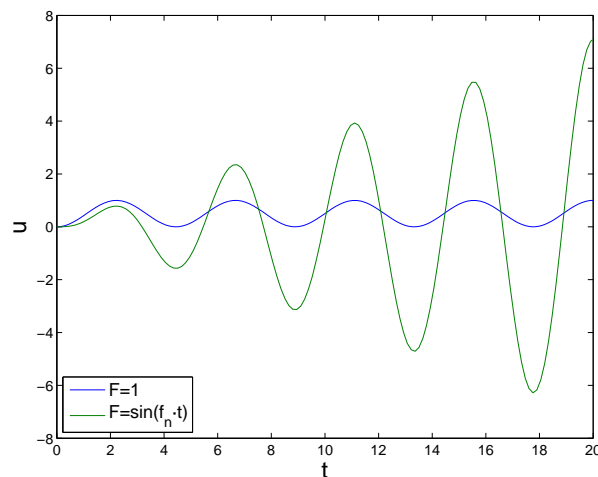


Figure 2

## Question 2

A uniform axial bar with length  $L$  and negligible mass which is clamped at both ends and has a weight of mass  $m$  placed at its center is subjected to three forces : The upper reaction  $R_2$  ( at  $y = L$ ), the reaction at the bottom  $R_1$  ( at  $y = 0$ ) and the force due to the mass placed at  $y = 0$  which is  $mg$ .  $R_1$  and  $R_2$  can be obtained from equilibrium of forces and compatibility condition:

$$\sum F = 0 \quad \rightarrow \quad mg = R_1 - R_2$$

$$u(0) = u(L) = 0 \quad \rightarrow \quad 0 = \frac{R_1 L}{EA} + \frac{R_1 L}{EA} \quad \rightarrow \quad R_1 = -R_2 = \frac{mg}{2}$$

The displacement at  $y = \frac{L}{2}$  is :

$$u_{\frac{L}{2}} = \frac{R_1 L}{EA} = \frac{mgL}{2EA}$$

The stiffness  $K$  is then:

$$K = \frac{mg}{u_{\frac{L}{2}}} = \frac{2EA}{L}$$

Since we are neglecting the mass of the bar, the total mass of the system is  $m$ . Thus, the natural frequency of vibration can be expressed as:

$$f_n = \sqrt{2 \frac{EA}{mL}}$$

### Question 3

The consistent element mass matrix is calculated as:

$$m = \int_{\Omega^e} \mathbf{N}^T \mathbf{N} \rho dV$$

Using an isoparametric representation of the two-node bar element:

$$\begin{aligned} m &= \rho A \int_{-1}^1 \begin{bmatrix} \frac{1}{2}(1-\xi) \\ \frac{1}{2}(1+\xi) \end{bmatrix} \begin{bmatrix} \frac{1}{2}(1-\xi) & \frac{1}{2}(1+\xi) \end{bmatrix} |J| d\xi = \\ &= \rho \frac{AL}{8} \int_{-1}^1 \begin{bmatrix} (1-\xi)^2 & (1-\xi)(1+\xi) \\ (1-\xi)(1+\xi) & (1+\xi)^2 \end{bmatrix} d\xi = \rho \frac{AL}{8} \int_{-1}^1 \begin{bmatrix} 1-2\xi+\xi^2 & 1-\xi^2 \\ 1-\xi^2 & 1+2\xi+\xi^2 \end{bmatrix} d\xi = \\ &= \rho \frac{AL}{8} \begin{bmatrix} \xi - \xi^2 + \frac{\xi^3}{3} & \xi - \frac{\xi^3}{3} \\ \xi - \frac{\xi^3}{3} & \xi + \xi^2 + \frac{\xi^3}{3} \end{bmatrix}_{-1}^1 = \rho \frac{AL}{8} \begin{bmatrix} \frac{8}{3} & \frac{4}{3} \\ \frac{4}{3} & \frac{8}{3} \end{bmatrix} = \rho \frac{AL}{3} \begin{bmatrix} 1 & \frac{1}{2} \\ \frac{1}{2} & 1 \end{bmatrix} = \boxed{\begin{bmatrix} \rho \frac{AL}{3} & \rho \frac{AL}{6} \\ \rho \frac{AL}{6} & \rho \frac{AL}{3} \end{bmatrix}} \end{aligned}$$

### Question 4

We can express the variation of area as:

$$A(\xi) = \sum_{i=1}^2 N_i(\xi) A_i = \frac{A_1}{2}(1-\xi) + \frac{A_2}{2}(1+\xi)$$

Thus:

$$\begin{aligned} m &= \int_{-1}^1 \mathbf{N}^T \mathbf{N} \rho A |J| d\xi = \rho \frac{L}{2} \int_{-1}^1 \begin{bmatrix} \frac{1}{2}(1-\xi) \\ \frac{1}{2}(1+\xi) \end{bmatrix} \begin{bmatrix} \frac{1}{2}(1-\xi) & \frac{1}{2}(1+\xi) \end{bmatrix} \left( \frac{A_1}{2}(1-\xi) + \frac{A_2}{2}(1+\xi) \right) d\xi = \\ &= \rho \frac{L}{16} \int_{-1}^1 \begin{bmatrix} (1-\xi)^2 & (1-\xi)(1+\xi) \\ (1-\xi)(1+\xi) & (1+\xi)^2 \end{bmatrix} (A_1(1-\xi) + A_2(1+\xi)) d\xi = \\ &= \rho \frac{L}{16} \int_{-1}^1 A_1 \begin{bmatrix} (1-\xi)^3 & (1-\xi)^2(1+\xi) \\ (1-\xi)^2(1+\xi) & (1+\xi)^2(1-\xi) \end{bmatrix} + A_2 \begin{bmatrix} (1-\xi)^2(1+\xi) & (1-\xi)(1+\xi)^2 \\ (1-\xi)(1+\xi)^2 & (1+\xi)^3 \end{bmatrix} d\xi = \end{aligned}$$

$$\begin{aligned}
&= \rho \frac{L}{16} \int_{-1}^1 A_1 \begin{bmatrix} 1 - 3\xi + 3\xi^2 - \xi^3 & 1 - \xi - \xi^2 + \xi^3 \\ 1 - \xi - \xi^2 + \xi^3 & 1 + \xi - \xi^2 - \xi^3 \end{bmatrix} + A_2 \begin{bmatrix} 1 - \xi - \xi^2 + \xi^3 & 1 + \xi - \xi^2 - \xi^3 \\ 1 + \xi - \xi^2 - \xi^3 & 1 + 3\xi + 3\xi^2 + 3\xi^3 \end{bmatrix} = \\
&= \rho \frac{L}{16} \left[ A_1 \begin{bmatrix} \xi - \frac{3\xi^2}{2} + \xi^3 - \frac{\xi^4}{4} & \xi - \frac{\xi^2}{2} - \frac{\xi^3}{3} + \frac{\xi^4}{4} \\ \xi - \frac{\xi^2}{2} - \frac{\xi^3}{3} + \frac{\xi^4}{4} & \xi + \frac{\xi^2}{2} - \frac{\xi^3}{3} - \frac{\xi^4}{4} \end{bmatrix} + A_2 \begin{bmatrix} \xi - \frac{\xi^2}{2} - \frac{\xi^3}{3} + \frac{\xi^4}{4} & \xi + \frac{\xi^2}{2} - \frac{\xi^3}{3} - \frac{\xi^4}{4} \\ \xi + \frac{\xi^2}{2} - \frac{\xi^3}{3} - \frac{\xi^4}{4} & \xi + \frac{3\xi^2}{2} + \xi^3 + \frac{\xi^4}{4} \end{bmatrix} \right]_{-1}^1 = \\
&= \boxed{\begin{bmatrix} \rho L \frac{3A_1+A_2}{12} & \rho L \frac{A_1+A_2}{12} \\ \rho L \frac{A_1+A_2}{12} & \rho L \frac{A_1+3A_2}{12} \end{bmatrix}}
\end{aligned}$$

## Question 5

The simplest procedure to obtain the lumped mass matrix of the 3D 2-node bar is to assign half of the mass of the bar to every node:

$$m = \frac{\rho LA}{2} \mathbf{I}_6 = \boxed{\begin{bmatrix} \frac{\rho LA}{2} & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{\rho LA}{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & \frac{\rho LA}{2} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{\rho LA}{2} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{\rho LA}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{\rho LA}{2} \end{bmatrix}}$$